

Modeling of a Pantograph Haptic Device

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Kinematics of the Pantograph Haptic Device

The pantograph haptic device consists of 5 links with lengths l_i for $i = 1, \dots, 5$. First, we assign frames of reference to each link, as shown in Fig. 1. The holonomic constrain of the pantograph mechanisms is given by

$$l_1 \mathbf{a}_1 + l_2 \mathbf{b}_1 - l_3 \mathbf{c}_1 - l_4 \mathbf{d}_1 - l_0 \mathbf{n}_1 = 0, \quad (1)$$

where l_i for $i = 1, \dots, 5$ is the length of links N, A, B, C, and D, respectively. We assume that link N is the frame of reference and calculate the rotation matrices of each link with respect to frame N as follows:

$${}^N \mathbf{R}^A = \begin{bmatrix} \cos q_1 & -\sin q_1 & 0 \\ \sin q_1 & \cos q_1 & 0 \\ 0 & 0 & 1 \end{bmatrix}, \quad (2)$$

where ${}^N \mathbf{R}^A$ is the rotation matrix of link A with respect to the frame of reference N, and q_1 is the generalized coordinate of the link A. Similarly, the rotation matrix (${}^N \mathbf{R}^B$) of link B with respect to the frame of reference N is given by

$${}^N \mathbf{R}^B = \begin{bmatrix} \cos q_2 & -\sin q_2 & 0 \\ \sin q_2 & \cos q_2 & 0 \\ 0 & 0 & 1 \end{bmatrix}, \quad (3)$$

where q_2 is the generalized coordinate of link B. Rotation matrices of bodies C (${}^N \mathbf{R}^C$) and D (${}^N \mathbf{R}^D$) with respect to N are given respectively using

$${}^N \mathbf{R}^C = \begin{bmatrix} \cos q_3 & -\sin q_3 & 0 \\ \sin q_3 & \cos q_3 & 0 \\ 0 & 0 & 1 \end{bmatrix} \quad \text{and} \quad {}^N \mathbf{R}^D = \begin{bmatrix} \cos q_4 & -\sin q_4 & 0 \\ \sin q_4 & \cos q_4 & 0 \\ 0 & 0 & 1 \end{bmatrix}, \quad (4)$$

where q_3 and q_4 are the generalized coordinates of links C and D, respectively. All rotations of our pantograph haptic device are in-plane. Therefore, the angular velocities of the links are calculated by inspection as follows:

$${}^N \omega^A = \dot{q}_1 \mathbf{n}_3, \quad (5)$$

where ${}^N \omega^A$ is the angular velocity of link A with respect to the frame of reference N, and \dot{q}_1 is the time-derivative of q_1 . Similarly, the angular velocities of links B, C, and D are given by

$${}^N \omega^B = \dot{q}_2 \mathbf{n}_3, \quad {}^N \omega^C = \dot{q}_3 \mathbf{n}_3, \quad \text{and} \quad {}^N \omega^D = \dot{q}_4 \mathbf{n}_3, \quad (6)$$

where ${}^N \omega^B$, ${}^N \omega^C$, and ${}^N \omega^D$ are the angular velocities of links B, C, and D with respect to the frame of reference N, respectively. \mathbf{n}_3 is a unit vector perpendicular to the page. Now we calculate the relation between the task space ($\mathbf{x} = (x \ y)^T$) and joint space ($\mathbf{q} = (q_1 \ q_2 \ q_3 \ q_4)^T$) of the pantograph haptic device, where x and y are the Cartesian coordinates of point e with respect to the frame of reference N, as shown in Fig. 1.

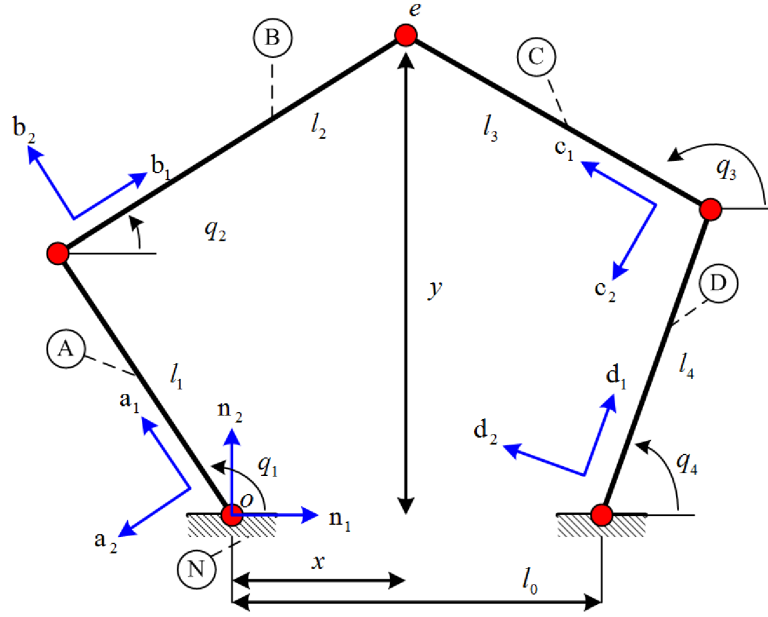


Figure 1. Kinematics of the pantograph haptic device. The position of the end-effector (point of interaction between the operator and the pantograph) is denoted using e , and is represented using position vector \mathbf{r}^{oe} .

0.1 Configuration Level Kinematics

In order to solve for x and y using q_1 and q_4 (the active angles of the pantograph haptic device), we define the following vector:

$$\mathbf{r}^{oe} = x\mathbf{n}_1 + y\mathbf{n}_2, \quad (7)$$

$$= l_1\mathbf{a}_1 + l_2\mathbf{b}_1, \quad (8)$$

$$= l_1(\cos q_1\mathbf{n}_1 + \sin q_1\mathbf{n}_2) + l_2(\cos q_2\mathbf{n}_1 + \sin q_2\mathbf{n}_2), \quad (9)$$

$$= (l_1 \cos q_1 + l_2 \cos q_2)\mathbf{n}_1 + (l_1 \sin q_1 + l_2 \sin q_2)\mathbf{n}_2, \quad (10)$$

where \mathbf{r}^{oe} is the position vector of point e in the frame of reference N . Using (10), we obtain the following 2 scalar equations:

$$x = l_1 \cos q_1 + l_2 \cos q_2, \quad (11)$$

$$y = l_1 \sin q_1 + l_2 \sin q_2. \quad (12)$$

Equations (11) and (12) represent the forward kinematics of the pantograph haptic device. Now we need to solve for q_2 in terms of q_1 using the holonomic constrain (1). This step is necessary since q_2 is a passive angle and has to be represented using the active angles of the device (q_1 and q_4). Using (1), we obtain

$$l_1(\cos q_1\mathbf{n}_1 + \sin q_1\mathbf{n}_2) + l_2(\cos q_2\mathbf{n}_1 + \sin q_2\mathbf{n}_2) - l_3(\cos q_3\mathbf{n}_1 + \sin q_3\mathbf{n}_2) - l_4(\cos q_4\mathbf{n}_1 + \sin q_4\mathbf{n}_2) - l_o\mathbf{n}_1 = 0, \quad (13)$$

Equation (13) provides the following two scalar equations:

$$l_1 \cos q_1 + l_2 \cos q_2 - l_3 \cos q_3 - l_4 \cos q_4 - l_o = 0, \quad (14)$$

$$l_1 \sin q_1 + l_2 \sin q_2 - l_3 \sin q_3 - l_4 \sin q_4 = 0. \quad (15)$$

Solving (14) and (15) using Newton-Raphson method. We make the following definitions:

$$\mathbf{f} = \begin{pmatrix} f_1 \\ f_2 \end{pmatrix} = \begin{pmatrix} l_1 \cos q_1 + l_2 \cos q_2 - l_3 \cos q_3 - l_4 \cos q_4 - l_o \\ l_1 \sin q_1 + l_2 \sin q_2 - l_3 \sin q_3 - l_4 \sin q_4 \end{pmatrix}. \quad (16)$$

All link length and q_1 and q_4 are know, therefore we solve for q_2 and q_3 . The passive angles are defined as, $\mathbf{q}_p = [q_2 \quad q_3]^T$. Using Newton-Raphson method, \mathbf{q}_p is calculated using

$$\mathbf{q}_p^{i+1} = \mathbf{q}_p^i - \frac{\delta \mathbf{f}}{\delta \mathbf{q}_p} \mathbf{f}. \quad (17)$$

Once q_2 is solved interns of q_1 and q_4 , the configuration level kinematics is completed.

0.2 Configuration Level Inverse Kinematics

Now we solve for q_1 and q_4 based on x and y of point e. We rewrite (11) and (12) as follows:

$$x - l_1 \cos q_1 - l_2 \cos q_2 = 0, \quad (18)$$

$$y - l_1 \sin q_1 - l_2 \sin q_2 = 0. \quad (19)$$

Recall the holonomic constrain and define the following scalar equations:

$$l_1 \cos q_1 + l_2 \cos q_2 - l_3 \cos q_3 - l_4 \cos q_4 - l_o = 0, \quad (20)$$

$$l_1 \sin q_1 + l_2 \sin q_2 - l_3 \sin q_3 - l_4 \sin q_4 = 0. \quad (21)$$

Solve (18), (19), (20), and (21) using Newton-Raphson.

0.3 Motion Level Forward Kinematics

Taking the time derivative of the position vector of point e \mathbf{r}^{oe} in the frame of reference N yields

$${}^N \frac{d}{dt} \mathbf{r}^{oe} = {}^N \mathbf{v}^e = \dot{x} \mathbf{n}_1 + \dot{y} \mathbf{n}_2 \quad (22)$$

$$= {}^N \frac{d}{dt} (l_1 \mathbf{a}_1 + l_2 \mathbf{b}_1) = l_1^N \frac{d}{dt} \mathbf{a}_1 + l_2^N \frac{d}{dt} \mathbf{b}_1 \quad (23)$$

$$= l_1 \left({}^A \frac{d}{dt} \mathbf{a}_1 + {}^N \omega^A \times \mathbf{a}_1 \right) + l_2 \left({}^B \frac{d}{dt} \mathbf{b}_1 + {}^N \omega^B \times \mathbf{b}_1 \right) \quad (24)$$

$$= l_1 \dot{q}_1 \mathbf{n}_3 \times \mathbf{a}_1 + l_2 \dot{q}_2 \mathbf{n}_3 \times \mathbf{b}_1 \quad (25)$$

$$= l_1 \dot{q}_1 \mathbf{a}_2 + l_2 \dot{q}_2 \mathbf{b}_2 \quad (26)$$

$$= l_1 \dot{q}_1 (-\sin q_1 \mathbf{n}_1 + \cos q_1 \mathbf{n}_2) + l_2 \dot{q}_2 (-\sin q_2 \mathbf{n}_1 + \cos q_2 \mathbf{n}_2) \quad (27)$$

Finally, the forward level kinematics is given by

$$\dot{x} = -l_1 \sin q_1 \dot{q}_1 - l_2 \sin q_2 \dot{q}_2, \quad (28)$$

$$\dot{y} = l_1 \cos q_1 \dot{q}_1 + l_2 \cos q_2 \dot{q}_2. \quad (29)$$

Arranging the terms in a linear form we obtain

$$\dot{\mathbf{x}} = \begin{bmatrix} \dot{x} \\ \dot{y} \end{bmatrix} = \begin{bmatrix} -l_1 \sin q_1 & -l_2 \sin q_2 \\ l_1 \cos q_1 & l_2 \cos q_2 \end{bmatrix} \begin{bmatrix} \dot{q}_1 \\ \dot{q}_2 \end{bmatrix} = \mathbf{J}(\mathbf{q}) \dot{\mathbf{q}}. \quad (30)$$

where $\mathbf{J}(\mathbf{q})$ is the Jacobian matrix of the pantograph haptic device.

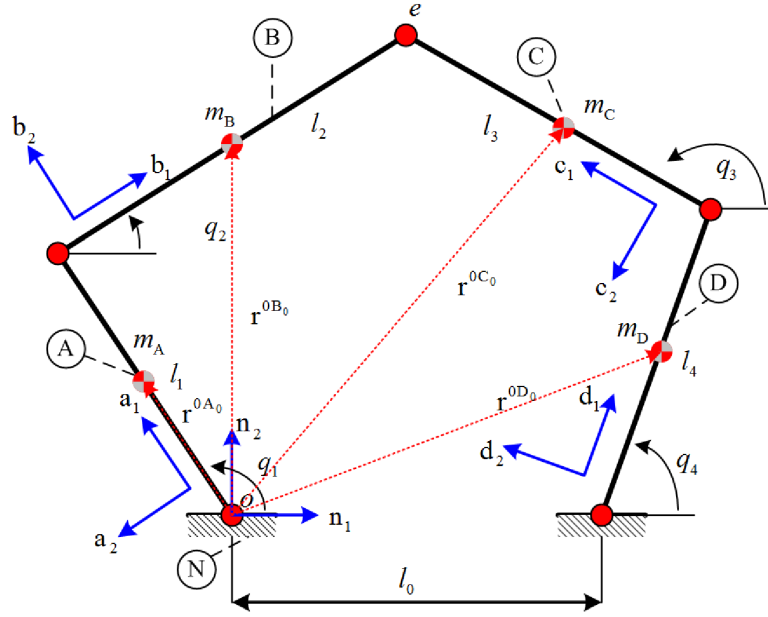


Figure 2. Dynamics of the pantograph haptic device. The red dashed vectors represent position vectors to the center of masses.

0.4 Closed-Form Solution of the Inverse Kinematics Problem

Newton-Raphson method is not recommended in implementation since it is an iterative method. Therefore, we devise the following closed-form solution of the inverse-kinematics:

$$q_1 = 2 \tan^{-1} \left(\frac{-A + \sqrt{A^2 + B_1^2 - C_1^2}}{C_1 - B_1} \right) \quad \text{and} \quad q_2 = 2 \tan^{-1} \left(\frac{-A - \sqrt{A^2 + B_2^2 - C_2^2}}{C_2 - B_2} \right), \quad (31)$$

where the variables A , B_1 , B_2 , C_1 , and C_2 are given by

$$A = -2l_2y, \quad (32)$$

$$B_1 = -2l_2 \left(x - \frac{l_0}{2} \right), \quad (33)$$

$$B_2 = -2l_2 \left(x + \frac{l_0}{2} \right), \quad (34)$$

$$C_1 = x^2 + y^2 + \left(\frac{l_0}{2} \right)^2 + l_2^2 - l_1^2 - l_0x, \quad (35)$$

$$C_2 = x^2 + y^2 + \left(\frac{l_0}{2} \right)^2 + l_2^2 - l_1^2 + l_0x. \quad (36)$$

1 Dynamics of the Pantograph Haptic Device

The position vector of the center of mass of link A is given by

$$\mathbf{r}^{0A_0} = \frac{l_1}{2} \mathbf{a}_1 = \frac{l_1}{2} \cos q_1 \mathbf{n}_1 + \frac{l_1}{2} \sin q_1 \mathbf{n}_2, \quad (37)$$

where \mathbf{r}^{0A_0} is the position vector of the center of mass of link A. The position vector of the center of mass of link B is given by

$$\mathbf{r}^{0B_0} = l_1 \mathbf{a}_1 + \frac{l_2}{2} \mathbf{b}_1, \quad (38)$$

$$= l_1 \cos q_1 \mathbf{n}_1 + l_1 \sin q_1 \mathbf{n}_2 + \frac{l_2}{2} \cos q_2 \mathbf{n}_1 + \frac{l_2}{2} \sin q_2 \mathbf{n}_2, \quad (39)$$

$$= \left(l_1 \cos q_1 + \frac{l_2}{2} \cos q_2 \right) \mathbf{n}_1 + \left(l_1 \sin q_1 + \frac{l_2}{2} \sin q_2 \right) \mathbf{n}_2. \quad (40)$$

Similarly, the position vector of the center of mass of link D is give by

$$\mathbf{r}^{0D_0} = l_0 \mathbf{n}_1 + \frac{l_4}{2} \mathbf{d}_1, \quad (41)$$

$$= l_0 \mathbf{n}_1 + \frac{l_4}{2} (\cos q_4 \mathbf{n}_1 + \sin q_4 \mathbf{n}_2), \quad (42)$$

$$= \left(l_0 + \frac{l_4}{2} \cos q_4 \right) \mathbf{n}_1 + \frac{l_4}{2} \sin q_4 \mathbf{n}_2. \quad (43)$$

Finally, the position vector of the center of mass of link C is given by

$$\mathbf{r}^{0C_0} = l_0 \mathbf{n}_1 + l_4 \mathbf{d}_1 + \frac{l_3}{2} \mathbf{c}_1, \quad (44)$$

$$= l_0 \mathbf{n}_1 + l_4 \cos q_4 \mathbf{n}_1 + l_4 \sin q_4 \mathbf{n}_2 + \frac{l_3}{2} \cos q_3 \mathbf{n}_1 + \frac{l_3}{2} \sin q_3 \mathbf{n}_2, \quad (45)$$

$$= \left(l_0 + l_4 \cos q_4 + \frac{l_3}{2} \cos q_3 \right) \mathbf{n}_1 + \left(l_4 \sin q_4 + \frac{l_3}{2} \sin q_3 \right) \mathbf{n}_2. \quad (46)$$

Now we calculate the velocity vector of the calculated vectors in the frame of reference N as follows:

$${}^N \mathbf{v}^{A_0} = {}^N \frac{d}{dt} \mathbf{r}^{0A_0} = -\frac{l_1}{2} \dot{q}_1 \sin q_1 \mathbf{n}_1 + \frac{l_1}{2} \dot{q}_1 \cos q_1 \mathbf{n}_2, \quad (47)$$

where ${}^N \mathbf{v}^{A_0}$ is the velocity vector of the center of mass of link A in the frame of reference N. The velocity vector of the center of mass of link B is calculated using

$${}^N \mathbf{v}^{B_0} = {}^N \frac{d}{dt} \mathbf{r}^{0B_0} = -\left(l_1 \dot{q}_1 \sin q_1 + \frac{l_2}{2} \dot{q}_2 \sin q_2 \right) \mathbf{n}_1 + \left(l_1 \dot{q}_1 \cos q_1 + \frac{l_2}{2} \dot{q}_2 \cos q_2 \right) \mathbf{n}_2, \quad (48)$$

where ${}^N \mathbf{v}^{B_0}$ is the velocity vector of the center of mass of link B in the frame of reference N. Similarly, the velocity vector of the center of mass of link D is given by

$${}^N \mathbf{v}^{D_0} = {}^N \frac{d}{dt} \mathbf{r}^{0D_0} = -\left(\frac{l_4}{2} \dot{q}_4 \sin q_4 \right) \mathbf{n}_1 + \left(\frac{l_4}{2} \dot{q}_4 \cos q_4 \right) \mathbf{n}_2, \quad (49)$$

where ${}^N \mathbf{v}^{D_0}$ is the velocity vector of the center of mass of link D in the frame of reference N. Finally, the velocity vector of the center of mass of link C is calculated as follows:

$${}^N \mathbf{v}^{C_0} = {}^N \frac{d}{dt} \mathbf{r}^{0C_0} = -\left(l_4 \dot{q}_4 \sin q_4 + \frac{l_3}{2} \dot{q}_3 \sin q_3 \right) \mathbf{n}_1 + \left(l_4 \dot{q}_4 \cos q_4 + \frac{l_3}{2} \dot{q}_3 \cos q_3 \right) \mathbf{n}_2, \quad (50)$$

where ${}^N \mathbf{v}^{C_0}$ is the velocity vector of the center of mass of link C in the frame of reference N.

Link A is in rotational motion only. Therefore, its kinetic energy (T_A) is calculated using

$$T_A = \frac{1}{2} \frac{m_A l_1^2}{3} \dot{q}_1^2, \quad (51)$$

Link D exhibits only rotational motion. Therefore, its kinetic energy (T_D) is calculated using

$$T_D = \frac{1}{2} \frac{m_D l_4^2}{3} \dot{q}_4^2. \quad (52)$$

The kinetic energy of Link B is due to translation and rotation and can be calculated as

$$T_B = \frac{1}{2} I_{CB} \dot{q}_2^2 + \frac{1}{2} m_B^N \mathbf{v}^{B_0} \cdot^N \mathbf{v}^{B_0}, \quad (53)$$

$$= \frac{1}{2} \frac{m_B l_2^2}{12} \dot{q}_2^2 + \frac{1}{2} m_B \left(l_1^2 \dot{q}_1^2 + \frac{l_2^2}{4} \dot{q}_2^2 + \frac{l_1 l_2}{2} \dot{q}_1 \dot{q}_2 \cos(q_1 - q_2) \right). \quad (54)$$

Similarly, the kinetic energy of Link C is due to translation and rotation and can be calculated as

$$T_C = \frac{1}{2} I_{CC} \dot{q}_3^2 + \frac{1}{2} m_C^N \mathbf{v}^{C_0} \cdot^N \mathbf{v}^{C_0}, \quad (55)$$

$$= \frac{1}{2} \frac{m_C l_3^2}{12} \dot{q}_3^2 + \frac{1}{2} m_C \left(l_4^2 \dot{q}_4^2 + \frac{l_3^2}{4} \dot{q}_3^2 + \frac{l_3 l_4}{2} \dot{q}_3 \dot{q}_4 \cos(q_3 - q_4) \right). \quad (56)$$

The total kinetic energy of the pantograph haptic device is

$$T = T_A + T_B + T_C + T_D. \quad (57)$$

Finally, the equations of motion of the pantograph haptic device are

$$\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{q}_i} \right) - \frac{\partial T}{\partial q_i} = Q_i \quad \text{for } i = 1, \dots, 4, \quad (58)$$

where Q_i is the generalized force associated with the generalized coordinate q_i .

The generalized force associated with q_1 is

$$Q_1 = \mathbf{F}_o \cdot \frac{\partial \mathbf{r}^{oe}}{\partial q_1} = -f_{ox} l_1 \sin q_1 + f_{oy} l_1 \cos q_1 + T_1, \quad (59)$$

where \mathbf{F}_o is the interaction force of the operator with the pantograph haptic device at a point e . Further, f_{ox} and f_{oy} are the components of \mathbf{F}_o in the frame of reference, along \mathbf{n}_1 and \mathbf{n}_2 , respectively. T_1 is the control torque input exerted on the generalized coordinate q_1 . The generalized force associated with q_2 is

$$Q_2 = \mathbf{F}_o \cdot \frac{\partial \mathbf{r}^{oe}}{\partial q_2} = -f_{ox} l_2 \sin q_2 + f_{oy} l_2 \cos q_2, \quad (60)$$

The generalized force associated with q_3 is

$$Q_3 = \mathbf{F}_o \cdot \frac{\partial \mathbf{r}^{oe}}{\partial q_3} = -f_{ox} l_3 \sin q_3 + f_{oy} l_3 \cos q_3, \quad (61)$$

The generalized force associated with q_4 is

$$Q_4 = \mathbf{F}_o \cdot \frac{\partial \mathbf{r}^{oe}}{\partial q_4} = -f_{ox} l_4 \sin q_4 + f_{oy} l_4 \cos q_4 + T_2. \quad (62)$$

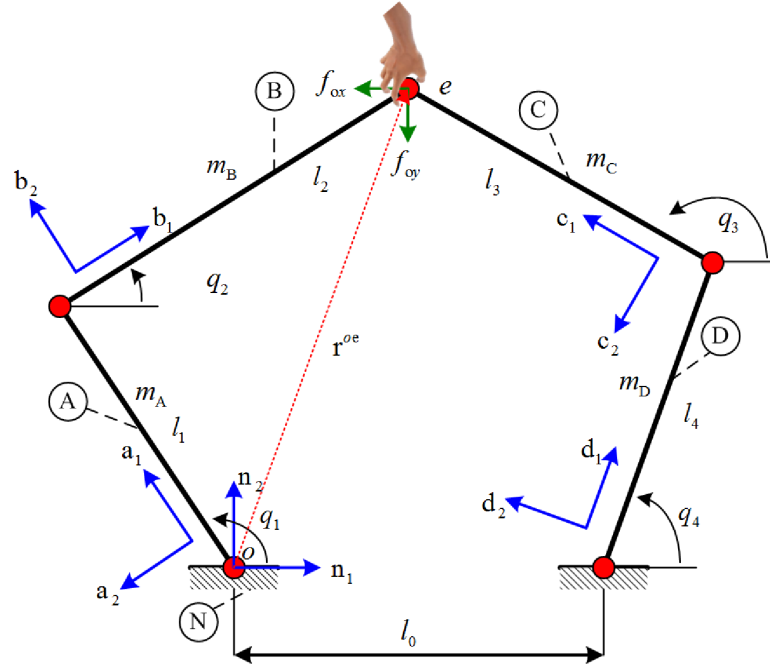


Figure 3. Generalized forces of the pantograph haptic device.

Now we recall (58) and determine the equations of motion of the pantograph haptic device as follows:

$$\begin{aligned} \left(\frac{m_A}{3} + m_B\right) l_1^2 \ddot{q}_1 + \frac{m_B l_1 l_2}{4} \ddot{q}_2 \cos(q_1 - q_2) - \frac{m_B l_1 l_2}{4} (\dot{q}_1 - \dot{q}_2) \dot{q}_2 \sin(q_1 - q_2) + \frac{m_B l_1 l_2}{4} \dot{q}_1 \dot{q}_2 \sin(q_1 - q_2) = \\ -f_{ox} l_1 \sin q_1 + f_{oy} l_1 \cos q_1 + T_1, \end{aligned} \quad (63)$$

$$\begin{aligned} \frac{m_B l_2^2}{3} \ddot{q}_2 + \frac{m_B l_1 l_2}{4} \ddot{q}_1 \cos(q_1 - q_2) - \frac{m_B l_1 l_2}{4} (\dot{q}_1 - \dot{q}_2) \dot{q}_1 \sin(q_1 - q_2) - \frac{m_B l_1 l_2}{4} \dot{q}_1 \dot{q}_2 \sin(q_1 - q_2) = \\ -f_{ox} l_2 \sin q_2 + f_{oy} l_2 \cos q_2, \end{aligned} \quad (64)$$

$$\begin{aligned} \frac{m_C l_3^2}{3} \ddot{q}_3 + \frac{m_C l_3 l_4}{4} \ddot{q}_4 \cos(q_3 - q_4) - \frac{m_C l_3 l_4}{4} (\dot{q}_3 - \dot{q}_4) \dot{q}_4 \sin(q_3 - q_4) - \frac{m_C l_3 l_4}{4} \dot{q}_3 \dot{q}_4 \sin(q_3 - q_4) = \\ -f_{ox} l_3 \sin q_3 + f_{oy} l_3 \cos q_3, \end{aligned} \quad (65)$$

$$\begin{aligned} \left(\frac{m_D}{3} + m_C\right) l_4^2 \ddot{q}_4 + \frac{m_C l_3 l_4}{4} \ddot{q}_3 \cos(q_3 - q_4) - \frac{m_C l_3 l_4}{4} (\dot{q}_3 - \dot{q}_4) \dot{q}_3 \sin(q_3 - q_4) + \frac{m_C l_3 l_4}{4} \dot{q}_3 \dot{q}_4 \sin(q_3 - q_4) = \\ -f_{ox} l_4 \sin q_4 + f_{oy} l_4 \cos q_4 + T_2, \end{aligned} \quad (66)$$

Based on (63), (64), (65), and (66), we define the following inertia and nonlinear damping matrices:

$$\mathbf{M}(\mathbf{q}) = \begin{bmatrix} \left(\frac{m_A}{3} + m_B\right) l_1^2 & \frac{m_B l_1 l_2}{4} \cos(q_1 - q_2) & 0 & 0 \\ \frac{m_B l_1 l_2}{4} \cos(q_1 - q_2) & \frac{m_B l_2^2}{3} & 0 & 0 \\ 0 & 0 & \frac{m_c l_3^2}{3} & \frac{m_c l_3 l_4}{4} \cos(q_3 - q_4) \\ 0 & 0 & \frac{m_c l_3 l_4}{4} \cos(q_3 - q_4) & \left(\frac{m_D}{3} + m_C\right) l_4^2 \end{bmatrix}, \quad (67)$$

where $\mathbf{M}(\mathbf{q})$ is the inertia matrix of the pantograph haptic device. Further, the nonlinear damping matrix is given by

$$\mathbf{D}(\mathbf{q}, \dot{\mathbf{q}}) = \begin{bmatrix} -\frac{m_B l_1 l_2}{4} (\dot{q}_1 - \dot{q}_2) \dot{q}_2 \sin(q_1 - q_2) + \frac{m_B l_1 l_2}{4} \dot{q}_1 \dot{q}_2 \sin(q_1 - q_2) \\ -\frac{m_B l_1 l_2}{4} (\dot{q}_1 - \dot{q}_2) \dot{q}_1 \sin(q_1 - q_2) - \frac{m_B l_1 l_2}{4} \dot{q}_1 \dot{q}_2 \sin(q_1 - q_2) \\ -\frac{m_c l_3 l_4}{4} (\dot{q}_3 - \dot{q}_4) \dot{q}_4 \sin(q_3 - q_4) - \frac{m_c l_3 l_4}{4} \dot{q}_3 \dot{q}_4 \sin(q_3 - q_4) \\ -\frac{m_c l_3 l_4}{4} (\dot{q}_3 - \dot{q}_4) \dot{q}_3 \sin(q_3 - q_4) + \frac{m_c l_3 l_4}{4} \dot{q}_3 \dot{q}_4 \sin(q_3 - q_4) \end{bmatrix} \quad (68)$$

$$= \underbrace{\begin{bmatrix} \frac{m_B l_1 l_2}{4} \dot{q}_2 s_{12} & -\frac{m_B l_1 l_2}{4} (\dot{q}_1 - \dot{q}_2) s_{12} & 0 & 0 \\ -\frac{m_B l_1 l_2}{4} (\dot{q}_1 - \dot{q}_2) s_{12} & -\frac{m_B l_1 l_2}{4} \dot{q}_1 s_{12} & 0 & 0 \\ 0 & 0 & \frac{m_c l_3 l_4}{4} \dot{q}_4 s_{34} & -\frac{m_c l_3 l_4}{4} (\dot{q}_3 - \dot{q}_4) s_{34} \\ 0 & 0 & -\frac{m_c l_3 l_4}{4} (\dot{q}_3 - \dot{q}_4) s_{34} & \frac{m_c l_3 l_4}{4} \dot{q}_3 s_{34} \end{bmatrix}}_{\mathbf{C}(\mathbf{q}, \dot{\mathbf{q}})} \begin{bmatrix} \dot{q}_1 \\ \dot{q}_2 \\ \dot{q}_3 \\ \dot{q}_4 \end{bmatrix} \quad (69)$$

Finally, the control torque input (\mathbf{T}) and interaction \mathbf{T}_h with the operator are defined as follows:

$$\mathbf{T} = \begin{bmatrix} T_1 \\ 0 \\ 0 \\ T_4 \end{bmatrix} \quad \text{and} \quad \mathbf{T}_h = \begin{bmatrix} -f_{ox} l_1 \sin q_1 + f_{oy} l_1 \cos q_1 \\ -f_{ox} l_2 \sin q_2 + f_{oy} l_2 \cos q_2 \\ -f_{ox} l_3 \sin q_3 + f_{oy} l_3 \cos q_3 \\ -f_{ox} l_4 \sin q_4 + f_{oy} l_4 \cos q_4 \end{bmatrix}. \quad (70)$$

Now the equation of motion of the pantograph haptic device can be written in the following compact form:

$$\mathbf{M}(\mathbf{q})\ddot{\mathbf{q}} + \mathbf{D}(\mathbf{q}, \dot{\mathbf{q}}) = \mathbf{T} + \mathbf{T}_h. \quad (71)$$

It can also be represented in the following form:

$$\mathbf{M}(\mathbf{q})\ddot{\mathbf{q}} + \mathbf{C}(\mathbf{q}, \dot{\mathbf{q}})\dot{\mathbf{q}} = \mathbf{T} + \mathbf{T}_h. \quad (72)$$

where $\mathbf{M}(\mathbf{q})$ is the inertia matrix ($\mathbf{M}(\mathbf{q})^T = \mathbf{M}(\mathbf{q}) > 0$). Further, $\mathbf{C}(\mathbf{q}, \dot{\mathbf{q}})\dot{\mathbf{q}}$ accounts for the centrifugal and Coriolis forces, and is related to the inertia matrix via, $\left(\dot{\mathbf{M}}(\mathbf{q}) - 2\mathbf{C}(\mathbf{q}, \dot{\mathbf{q}})\right)^T = -\left(\dot{\mathbf{M}}(\mathbf{q}) - 2\mathbf{C}(\mathbf{q}, \dot{\mathbf{q}})\right)$.

References

1. I. S. M. Khalil, E. Globovic, and A. Sabanovic, "High precision motion control of parallel robots with imperfections and manufacturing tolerances", in *Proceedings of the IEEE International Conference on Mechatronics (ICM)*, Istanbul, Turkey, pages 39-44, April 2011.
2. I. S. M. Khalil, V. Magdanz, S. O. Sanchez, O. G. Schmidt, and S. Misra, "Biocompatible, accurate, and fully autonomous: A sperm-driven micro-bio-robot", *Journal of Micro-Bio Robotics*, vol. 9, no. 3-4, pp. 79-86, August 2014.
3. K. Youakim, M. Ehab, O. Hatem, S. Misra, and I. S. M. Khalil, "Paramagnetic microparticles sliding on a surface: characterization and closed-loop motion control", in *Proceedings of the IEEE International Conference on Robotics and Automation (ICRA)*, Seattle, USA, pages 4068-4073, May-June 2015.

4. I. S. M. Khalil, B. E. Wissa, B. G. Salama, and S. Stramigioli, "Wireless Motion Control of Paramagnetic Microparticles using a Magnetic-Based Robotic System with an Open-Configuration," in *Proceedings of the International Conference on Manipulation, Manufacturing and Measurement on the Nanoscale (3M Nano)*, pp. 190-196, Changchun, China, October 2015.